

On the Mechanics of Axially Moving Materials

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TIEJ601 Postgraduate seminar (11th Jan 2010)

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About the Thesis 1/2

Title and content

- *On Numerical Simulation of Papermaking Processes*
- Content related to dynamics and instability analysis of axially moving materials

Supervisors

- Professor Raino Mäkinen
- Professor Pekka Neittaanmäki
- Visiting professor Nikolay Banichuk

About the Thesis 2/2

Status

- Started in spring term 2008
- Research underway: two papers published, two more being finished; more planned
- Will be completed in 2011

This talk

- Is focused on a general introduction for a postgraduate audience coming from various backgrounds.
- Some examples of results obtained so far will be shown, and
- Some possible directions for future research will be listed.

Research Status

N. Banichuk, J. Jeronen, P. Neittaanmäki, T. Tuovinen:

- On the Instability of an Axially Moving Elastic Plate. [1]
International Journal of Solids and Structures 47 (2010), pp. 91-99.
DOI: 10.1016/j.ijsolstr.2009.09.020.
- On the Instability of an Axially Moving Orthotropic Plate (working title).
Manuscript in preparation.
- Static Instability Analysis for Travelling Membranes and Plates
Interacting with Axially Moving Ideal Fluid. [2]
Journal of Fluids and Structures, accepted (to appear in 2010)
doi:10.1016/j.jfluidstructs.2009.09.006.
- Dynamical Behaviour of an Axially Moving Plate Undergoing Small
Cylindrical Deformation Submerged in Axially Flowing Ideal Fluid.
Manuscript in preparation.

What is a cat? 1/2

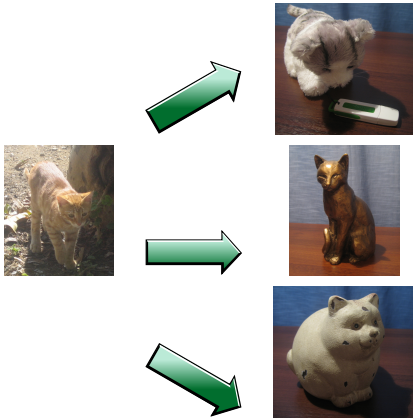


Photo taken by author.

How to model?

- Four paws, a set of claws, two ears, and a tail?
- An entire library filled with detailed observations of etology and biology?
- The only complete description of a cat is the cat itself.
(1. Ekeland [3]; same book also in English [4])

What is a cat? 2/2

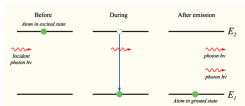
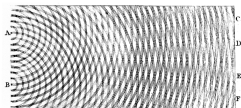


All images by author.

Modeling

- Simplifies in the desired way,
- While (hopefully) retaining the relevant aspects of the original system.
- The definition of “relevant” depends on the purpose of the model.

Models and Reality



Images from <http://en.wikipedia.org/wiki/Photon>

First two images PD, third © Wikipedia user DrBob and used under CC-by-SA-3.0.

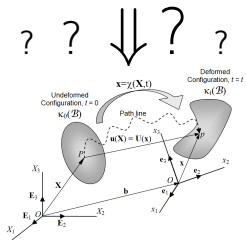
Models and reality

- Reality is fundamental
- Models are constructed to describe reality
- Comparison with reality determines model viability

Example

- Light: waves or particles?
- Correct answer: Neither!
- But can be modeled by both.

Why Continuum? 1/2



Why indeed?

- Isn't quantum mechanics a better model?

Well, it is true that

- Matter is, roughly speaking, made of discrete particles
- Counterintuitive QM effects such as tunneling and entanglement have been experimentally verified

Images from <http://en.wikipedia.org/wiki/Solid> and http://en.wikipedia.org/wiki/Continuum_mechanics

Images used under CC-by-SA-3.0. First © Thierry Dugnolle, second © Wikipedia user Sanpaz.

Why Continuum? 2/2

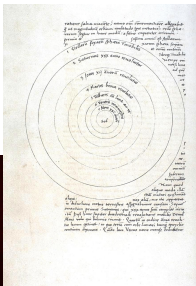
However, continuum models are fine, because

- QM phenomena are subtle
 - Macroscale implications possible, but require very precise engineering
 - E.g. only certain materials will “lase”
 - QM effects went undetected for centuries of physics!

Continuum models were built in the first place, because

- Reality does not care whether a model gets every detail “right”
- The continuum is a pretty good approximation under most circumstances.

Something to Keep in Mind



Getting the right idea is not always that simple

- What would a geocentric universe look like to the observers?
(R. Dawkins [5], original version attributed to L. Wittgenstein)

Images PD, from http://en.wikipedia.org/wiki/Geocentric_system

and http://en.wikipedia.org/wiki/De_revolutionibus_orbium_coelestium

Mathematical Modeling!

$$\begin{aligned}
 \frac{\frac{\partial^2 w}{\partial x^2} T}{1 + \left(\frac{\partial w}{\partial x}\right)^2} &= \Delta p \sqrt{1 + \left(\frac{\partial w}{\partial x}\right)^2} + mg + \frac{Eh^3}{12} \frac{\partial^4 w}{\partial x^4} \sqrt{1 + \left(\frac{\partial w}{\partial x}\right)^2} \\
 &+ \frac{m}{1 + \left(\frac{\partial w}{\partial x}\right)^2} \left\{ \left[1 + \left(\frac{\partial w}{\partial x}\right)^2 \right] \frac{\partial^2 w}{\partial t^2} + 2 \left[v \sqrt{1 + \left(\frac{\partial w}{\partial x}\right)^2} - \frac{\partial w}{\partial x} \frac{\partial w}{\partial t} \right] \frac{\partial^2 w}{\partial x \partial t} \right. \\
 &+ \left. \left[v^2 - \frac{2v \frac{\partial w}{\partial x} \frac{\partial w}{\partial t}}{\sqrt{1 + \left(\frac{\partial w}{\partial x}\right)^2}} + \frac{\left(\frac{\partial w}{\partial x}\right)^2 \left(\frac{\partial w}{\partial t}\right)^2}{1 + \left(\frac{\partial w}{\partial x}\right)^2} \right] \frac{\partial^2 w}{\partial x^2} \right\} \\
 &- \frac{1}{2} \frac{C_d \rho a}{\left[1 + \left(\frac{\partial w}{\partial x}\right)^2 \right]^3} \left\{ 2v \frac{\partial w}{\partial x} \frac{\partial w}{\partial t} \left[1 + \left(\frac{\partial w}{\partial x}\right)^2 \right]^{5/2} + \left(\frac{\partial w}{\partial t}\right)^2 \left[1 - \left(\frac{\partial w}{\partial x}\right)^4 \right] \right\}
 \end{aligned}$$

Source: [6]

About Mathematical Modeling

Mathematical Modeling

- Some phenomena are inherently more complex than others
 - Extreme example: turbulent flow vs. harmonic oscillator
 - Much of the simple physics has been done over the last few centuries
 - Complex problems can be solved numerically
 - ⇒ scientific computing
- Modeling of complex phenomena involves a tradeoff between analyzability and accuracy
 - Even then, analytical approaches may be somewhat useful
 - Design decisions made here determine the domain of applicability of the model

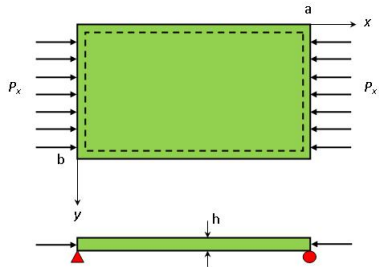
Plates?



Image from [http://en.wikipedia.org/wiki/Plate_\(dishware\)](http://en.wikipedia.org/wiki/Plate_(dishware))

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Plates!



Images from <http://en.wikipedia.org/wiki/File:ManilaPaper.jpg>
and http://commons.wikimedia.org/wiki/File:Rec_plate.jpg
First image © Nathan Beach, used under CC-by-SA-2.5, second image PD

What is an Elastic Plate? 1/2

Elastic plate

- An elastic, relatively flat piece of solid material of constant thickness; usually thin
- Idea: can abstract away the third dimension, handling it via special mathematical considerations
- Elastic behavior = reversible deformation
- Usually investigated phenomenon: **bending**

What is an Elastic Plate? 2/2

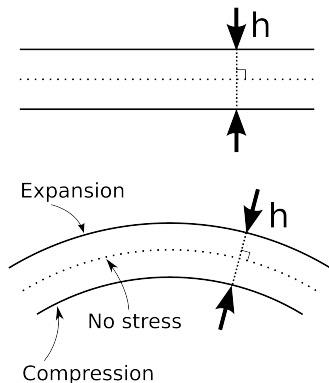


Image by author.

Kirchhoff (classical) plate theory
(see e.g. [7])

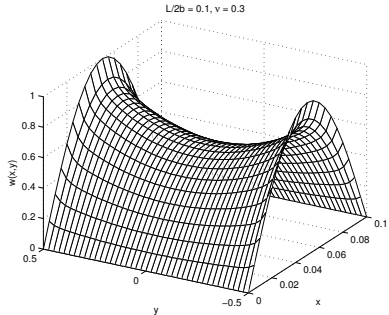
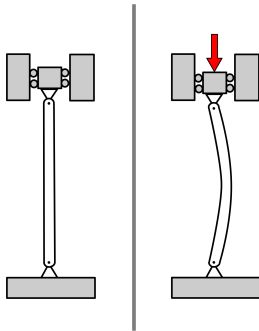
- No stresses in the middle plane of the plate
- Normals remain straight, and normal, during plate bending
- Normals are inextensible during plate bending (normal stresses can be disregarded)

Elastic Instability 1/2

Elastic instability

- *Buckling* i.e. static instability
- First analyzed for compressive loading of beam columns by L. Euler in the 18th century
- As load is slowly increased from zero, at first no transverse displacement
- At some critical load, the structure suddenly gives in
- May be relatively harmless or completely destructive
 - Buckled columns can carry load, if not increased further
 - Buckled paper web rips itself apart

Elastic Instability 2/2



First image PD, from <http://en.wikipedia.org/wiki/Buckling>
 Second image by author.

Stationary Structures vs. Moving Materials 1/2

Classical structural mechanics (and classical aeroelastic analysis)

- Concentrates on stationary, elastic structures (subjected to wind)
- Focuses on finding out how the structure deforms
- E.g. structural safety analysis for design of bridges

Note

- Airplanes are stationary in this sense!
- An airplane in flight behaves the same as a stationary one placed in a wind tunnel.

Stationary Structures vs. Moving Materials 2/2

Moving materials

- For phenomena where the material itself flows
- No clearly delimited physical object; instead, a continuous stream of material
- New material constantly enters one end of the analysis *domain*, and exits it at the other end
- Thus: different case; we cannot “ride” the material particles
⇒ Coriolis and centrifugal effects seen in a stationary coordinate system

Motivation 1/2



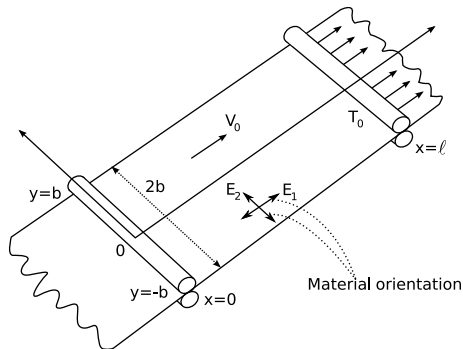
Figure: A paper web being driven through a paper machine.

Image under free license, from <http://en.wikipedia.org/wiki/Papermachine>

Motivation 2/2

Open draws in a paper machine

- Intervals, where the paper web travels with no mechanical support



Physical Models

Simple model

- Thin, elastic plate
 - Isotropic: elastic properties the same in every direction
 - Fully elastic: no irreversible (permanent) deformations

More accurate models

- Orthotropic elastic plate
 - Due to the fiber structure, the elastic properties of paper are different in different directions
- Viscoelasticity
 - Paper has also viscous properties \Rightarrow creep and relaxation
- For more, see the review article by Alava and Niskanen [8]

Elastics 1/2

Elastics

- Even linear elastics is geometrically nonlinear
- Can the nonlinearity be approximated away? The solution is...

Small deformation theory / Linearized theory

- Small out-of-plane (transverse) displacement
- Small angles between the plate and the coordinate axes
- One linear partial differential equation (PDE) is enough

And the point is?

- Why use a linear theory?

Elastics 2/2

Small deformation theory / Linearized theory

- Small out-of-plane (transverse) displacement
- Small angles between the plate and the coordinate axes
- One linear partial differential equation (PDE) is enough

This simplified model is

- Analyzable
 - Allows finding fundamental explanations for phenomena
- Understandable
 - The effects of the different parts of the theory can be known in detail
- Accurate enough in certain cases (instability analysis)
- A standard academic research topic

The Physical Model Used

Linear model (3D)

- Small deformation theory for thin, elastic plates
- Belongs to classical mechanics
- Based on linear PDEs

$$\begin{aligned}
 & m \frac{\partial^2 w}{\partial t^2} + 2mV_0 \frac{\partial^2 w}{\partial x \partial t} + mV_0^2 \frac{\partial^2 w}{\partial x^2} \\
 &= T_{xx} \frac{\partial^2 w}{\partial x^2} + 2T_{xy} \frac{\partial^2 w}{\partial x \partial y} + T_{yy} \frac{\partial^2 w}{\partial y^2} \\
 &\quad - D_1 \frac{\partial^4 w}{\partial x^4} - 2D_3 \frac{\partial^4 w}{\partial x^2 \partial y^2} - D_2 \frac{\partial^4 w}{\partial y^4} + q_f \quad \text{in } \Omega = [0, \ell] \times [-b, b]
 \end{aligned}$$

Physical Setup (3D)

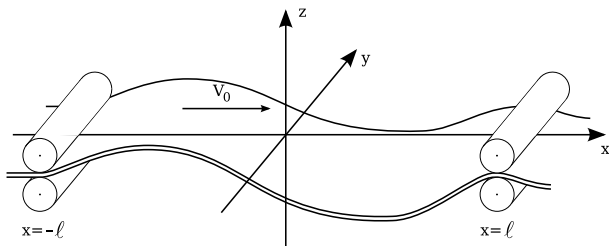


Figure: A model of an open draw (3D).

Phenomena (3D)

Investigated physical process

- Elastic deformations of a thin, axially moving plate (in vacuum)

Elastic Problem

- The investigated physical system is inherently unstable.
- Linear theory is applicable up to the first (critical) instability [9].
- The critical instability is of the static type [1].

Physical Setup (2D)

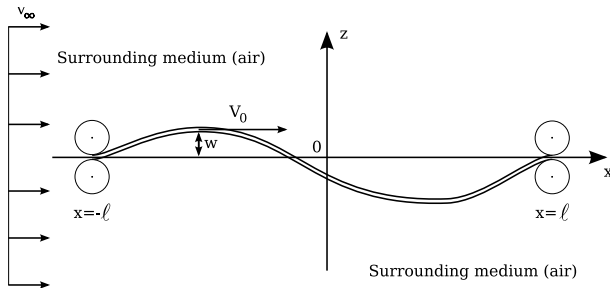


Figure: A model of an open draw (2D).

- The two-dimensional model can be used to investigate cylindrical deformations.

Phenomena (2D) 1/3

Investigated physical process

- Elastic, cylindrical deformations of a thin, axially moving plate, accounting for the effect of the surrounding air

Typically investigated cases

- Moving material surrounded by stationary air
 - Stationary structure subjected to axial flow
-
- In the paper [2] we combined these two cases.

Phenomena (2D) 2/3

Aeroelastic problem

- Vibrations coupled with the effect of the surrounding air
 - Contains the elastic problem
 - The presence of a surrounding medium changes the eigenfrequencies drastically [10], [11, 12].
- The instability of the system is still of the static type, but...
- By applying a dynamic analysis one can extract the eigenfrequencies, too.

Phenomena (2D) 3/3

Aeroelastic problem

- Closely connected to the problems of flow inside a pipe, and of a stationary structure subjected to axial flow [13].
- Results from stationary structures are not necessarily directly applicable [14], [15].
 - We found [2] that the assumption of cylindrical deformation only works well in case of narrow strips; cf. the stationary case where this approximation works best for wide plates [7].
- Ignoring viscosity may have a drastic effect on the critical velocities, but the correct behavior is obtained for the eigenfrequencies [15].

Numerical Results 1/8

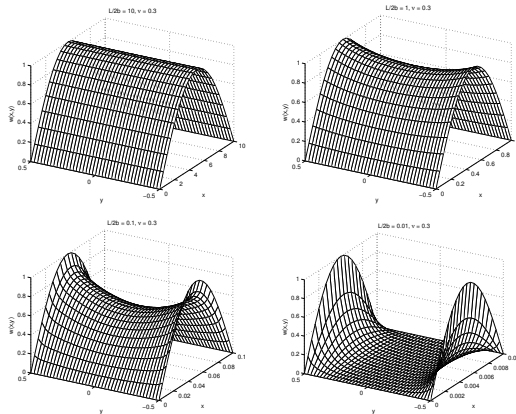
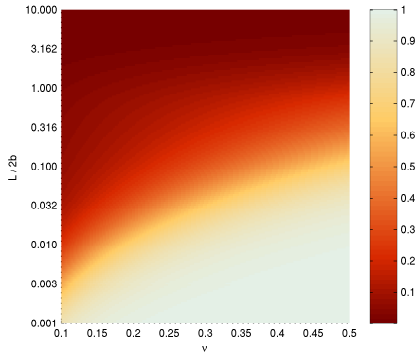


Figure: Buckling modes of isotropic plates.

Numerical Results 2/8



The degree of localization

of the buckling mode for isotropic plates, as a function of the aspect ratio $L/2b$ and the Poisson ratio ν . In the localized modes, most of the deformation occurs near the free boundaries.

In the figure, the color represents the relative degree of localization.

Numerical Results 3/8

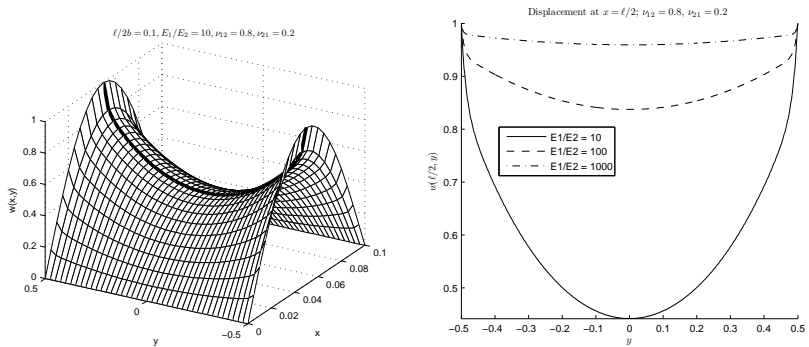
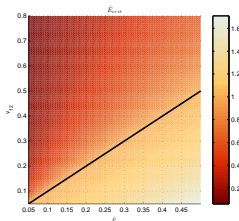
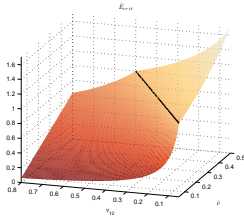


Figure: Buckling mode for dry paper. Orthotropic model, fibers along the machine direction.

Numerical Results 4/8



Parameter range where bending force stabilization is guaranteed for orthotropic plates. If a given parameter triplet \hat{E} , $\hat{\nu}$ and ν_{12} indicates a point on or below the surface in the figure, the bending forces will have a stabilizing effect regardless of the transverse displacement w . The parameters

$$\hat{E} \equiv \sqrt{E_2/E_1} \quad \text{and}$$

$$\hat{\nu} \equiv \sqrt{\nu_{12}\nu_{21}} .$$

Analytical Result

Aerodyn. pressure difference q_f as a functional of displacement w

$$q_f(x, t) = -\rho_f \left(\frac{1}{\tau} \frac{\partial}{\partial t} + \frac{1}{\ell} (v_\infty - V_0) \frac{\partial}{\partial x} \right) \dots$$

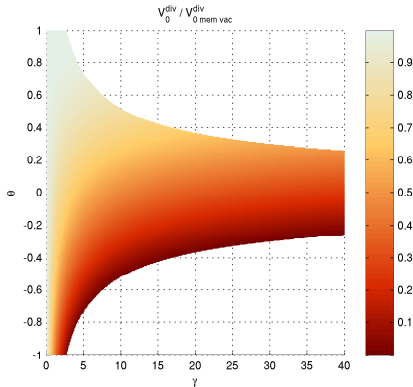
$$\dots \int_{-1}^1 N(\xi, x) \left(\frac{\ell}{\tau} \frac{\partial w}{\partial t} + (v_\infty - V_0) \frac{\partial w}{\partial x} \right) d\xi \quad (1)$$

where

$$N(\xi, x) \equiv \frac{1}{\pi} \ln \left| \frac{1 + \Lambda}{1 - \Lambda} \right|, \quad (2)$$

$$\Lambda(\xi, x) \equiv \left[\frac{(1-x)(1+\xi)}{(1-\xi)(1+x)} \right]^{1/2}. \quad (3)$$

Numerical Results 5/8



Dimensionless critical velocity

for an ideal membrane ($D = 0$), as a function of dimensionless fluid density

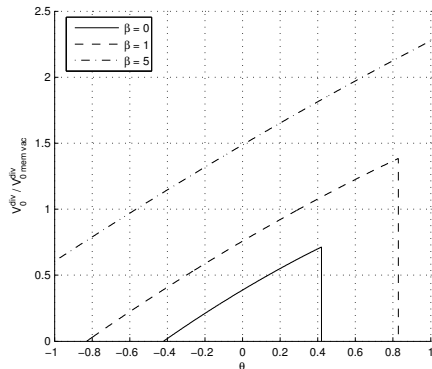
$$\gamma \equiv \frac{\ell \rho_f}{m}$$

and dimensionless fluid velocity

$$\theta \equiv \frac{v_\infty}{V_0^{\text{div mem vac}}},$$

where $V_0^{\text{div mem vac}} \equiv \sqrt{T/m}$.

Numerical Results 6/8



The effect of fluid velocity

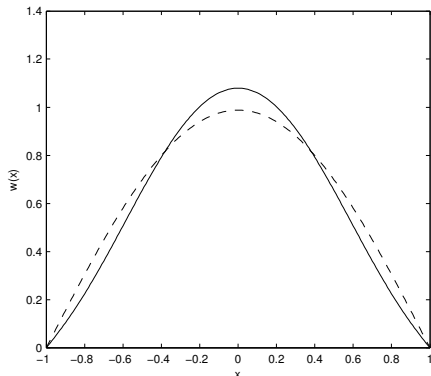
on the critical velocity of a plate,
 parametrized by dimensionless
 bending rigidity

$$\beta \equiv \frac{D}{\ell^2 T}.$$

In all cases in the figure, the
 dimensionless fluid density is

$$\gamma \equiv \frac{\ell \rho_f}{m} = 15.625.$$

Numerical Results 7/8



Buckling mode

for the parameter values

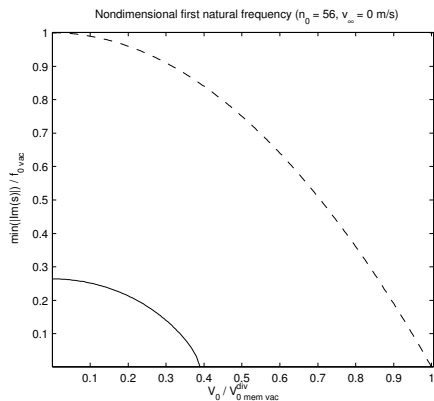
$$\beta \equiv \frac{D}{\ell^2 T} = 0.01 ,$$

$$\gamma \equiv \frac{\ell \rho_f}{m} = 15.625 , \quad \text{and}$$

$$\theta \equiv \frac{v_\infty}{V_{0 \text{ mem vac}}^{\text{div}}} = 0.43238 .$$

Dashed line: vacuum case for comparison.

Numerical Results 8/8



Lowest nondim. eigenfrequency

for an ideal membrane as a function of dimensionless membrane velocity

$$\lambda \equiv \frac{V_0}{V_{0 \text{ mem vac div}}^{\text{div}}},$$

where

$$V_{0 \text{ mem vac div}}^{\text{div}} \equiv \sqrt{T/m}.$$

Dashed line: vacuum case.

Solid line: in stationary ideal fluid.

Directions for Future Research 1/2

Directions for future research

- Currently planned:
 - Non-homogeneous tension distribution at supports
 - Linear tension profile (preliminary work begun)
 - Nonlinear tension profile
 - Effects of viscoelasticity on the dynamics and instability
 - Finding a balance between structural integrity and stability
 - Small cracks may cause the whole web to break if the tension is too high

Directions for Future Research 2/2

Directions for future research

- Possible additional topics:
 - A posteriori error estimates for the simplest cases?
 - Numerical fluid-structure interaction using more realistic fluid models (Navier–Stokes)?

Summary

- When modeling complex phenomena, there is a compromise between analyzability and completeness.
- Sometimes simple models are accurate enough, but on the other hand their applicability is limited.
- In the thesis, linearized Kirchhoff plate theory has been used as a basis for analytical and numerical research on mechanical instability phenomena in papermaking.

Thank you for your attention!



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